

HYDRONIC FORMULAS & PUMP LAWS

PUMP LAW #1 – FLOW vs. SPEED

Note: RPM, FPM, CFM, GPM, Frequency (Hz) and Dia. (Sheave/Impeller) are all interchangeable.

$$\frac{RPM_2}{RPM_1} = \frac{GPM_2}{GPM_1} \qquad RPM_2 = RPM_1 \times \left(\frac{GPM_2}{GPM_1} \right) \qquad GPM_2 = GPM_1 \times \left(\frac{RPM_2}{RPM_1} \right)$$

PUMP LAW #2 – FLOW vs. PRESSURE (ΔP , ΔH , SP , & Density)

$$\frac{\Delta P_2}{\Delta P_1} = \left(\frac{GPM_2}{GPM_1} \right)^2 \qquad \Delta P_2 = \Delta P_1 \times \left(\frac{GPM_2}{GPM_1} \right)^2 \qquad GPM_2 = GPM_1 \times \sqrt{\frac{\Delta P_2}{\Delta P_1}}$$

PUMP LAW #3 – FLOW vs. ENERGY (BHP & Amps)

$$\frac{BHP_2}{BHP_1} = \left(\frac{GPM_2}{GPM_1} \right)^3 \qquad BHP_2 = BHP_1 \times \left(\frac{GPM_2}{GPM_1} \right)^3 \qquad GPM_2 = GPM_1 \times \sqrt[3]{\frac{BHP_2}{BHP_1}}$$

STATIC EFFICIENCY: $PumpBHP = \frac{GPM \times TDH}{3960 \times Eff}$ $PumpEff = \frac{GPM \times TDH}{3960 \times BHP}$

HEAD:

Total Dynamic Head = Discharge_{TH} – Suction_{TH}

Total Dynamic Head = (Discharge_{SH} + Discharge_{VH}) – (Suction_{SH} + Suction_{VH})

NPSHA = Inlet_{PSIG} + Inlet VP_{PSI} + 14.7_{PSI} – Vapor Pressure_{PSI}

FLOW STATIONS:

$$GPM = C_v \times \sqrt[2]{\Delta P_{PSI}} \qquad C_v = \frac{GPM}{\sqrt[2]{\Delta P_{PSI}}} \qquad \Delta P_{PSI} = \left(\frac{GPM}{C_v} \right)^2$$

COOLING TOWERS:

Tower Approach = LWT – EAT_{wb}

Tower Range = EWT – LWT

$$\text{Coefficient of Performance} = \frac{BTUH}{Watts \times 3.41}$$

HEAT TRANSFER

WATER:

$$GPM = \frac{BTUH}{500 \times \Delta T} \qquad \Delta T = \frac{BTUH}{500 \times GPM} \qquad MBH = \frac{BTUH}{1000}$$

$$BTUH = GPM \times 500 \times \Delta T$$