

AIRFLOW FORMULAS & FAN LAWS

$$CFM = A \times V$$

$$CFM = Ak \times V$$

$$\frac{CFM}{V} = A$$

$$\frac{CFM}{A} = V$$

$$TP = SP + VP$$

$$TP - SP = VP$$

$$TP - VP = SP$$

$$VP = \left(\frac{V}{4005} \right)^2$$

$$V = 4005 \times \sqrt[2]{VP}$$

$$V = 1096.7 \times \sqrt[2]{\frac{VP}{Den}}$$

$$Den = .075 \times \left(\frac{530}{460 + T} \right) \times \left(\frac{BAR}{29.92} \right)$$

$$Fan_{SP} = Discharge_{SP} - Inlet_{TP}$$

FAN LAW #1 – FLOW vs. SPEED

Note: RPM, FPM, CFM, GPM, Frequency (Hz), and Dia. (Sheave/Impeller) are all interchangeable.

$$\frac{RPM_2}{RPM_1} = \frac{CFM_2}{CFM_1}$$

$$RPM_2 = RPM_1 \times \left(\frac{CFM_2}{CFM_1} \right)$$

$$CFM_2 = CFM_1 \times \left(\frac{RPM_2}{RPM_1} \right)$$

FAN LAW #2 – FLOW vs. PRESSURE (SP, ΔP, & Density)

$$\frac{SP_2}{SP_1} = \left(\frac{CFM_2}{CFM_1} \right)^2$$

$$SP_2 = SP_1 \times \left(\frac{CFM_2}{CFM_1} \right)^2$$

$$CFM_2 = CFM_1 \times \sqrt[2]{\frac{SP_2}{SP_1}}$$

FAN LAW #3 – FLOW vs. ENERGY (BHP & Amps)

$$\frac{BHP_2}{BHP_1} = \left(\frac{CFM_2}{CFM_1} \right)^3$$

$$BHP_2 = BHP_1 \times \left(\frac{CFM_2}{CFM_1} \right)^3$$

$$CFM_2 = CFM_1 \times \sqrt[3]{\frac{BHP_2}{BHP_1}}$$

STATIC EFFICIENCY:

$$FanBHP = \frac{CFM \times SP}{6356 \times SE}$$

$$FanSE = \frac{CFM \times SP}{6356 \times BHP}$$

AIR CHANGES:

$$ACH = \frac{CFM \times 60}{Room\ Volume}$$

$$CFM = Room\ Volume \times \left(\frac{ACH}{60} \right)$$

PERCENT OF AIR: (T=Temp, A=Air, M=Mixed, R=Return, O=Outside) *Used when there is at least a 20°ΔT between RA & OA. **CO₂ PPM can be substituted for Temperature.

$$\%OA = \frac{(RAT - MAT)}{(RAT - OAT)} \times 100$$

$$MAT = \frac{(\%OA \times OAT) + (\%RA \times RAT)}{100}$$

$$OAT = \frac{(MAT \times 100) - (\%RA \times RAT)}{\%OA}$$

$$RAT = \frac{(MAT \times 100) - (\%OA \times OAT)}{\%RA}$$

FAN TIP SPEED:

$$\text{Circ. (in.)} = \frac{\text{TS(FPM)} \times 12}{\text{RPM}} \quad \text{TS(FPM)} = \text{RPM} \times \frac{\text{Circ. (in.)}}{12} \quad \text{RPM} = \frac{\text{TS(FPM)} \times 12}{\text{Circ. (in.)}}$$

AIRFLOW STATIONS:

$$\text{CFM} = C_v \times \sqrt[2]{\Delta P_{(\text{in.})\text{H}_2\text{O}}} \quad C_v = \frac{\text{CFM}}{\sqrt[2]{\Delta P_{(\text{in.})\text{H}_2\text{O}}}} \quad \Delta P_{(\text{in.})\text{H}_2\text{O}} = \left(\frac{\text{CFM}}{C_v} \right)^2$$

BELTS & SHEAVES:

BL=Belt Length C=Centerline distance between shafts D=Fan Sheave dia. d=Motor Sheave dia.

$$\text{BL} = 2C + (1.57 \times (D + d)) + \frac{(D - d)^2}{4C}$$

$$\text{FanRPM} = \text{MotorRPM} \times \left(\frac{d}{D} \right)$$

$$\text{MotorRPM} = \text{FanRPM} \times \left(\frac{D}{d} \right)$$

$$\text{New Motor Sheave: } \text{dia}_2 = \text{dia}_1 \left(\frac{\text{RPM}_2}{\text{RPM}_1} \right)$$

$$\text{New Fan Sheave: } \text{Dia}_2 = \text{Dia}_1 \left(\frac{\text{RPM}_1}{\text{RPM}_2} \right)$$

$$\text{Fan}_{\text{Sp}} = \text{Discharge}_{\text{Sp}} - \text{Inlet}_{\text{Tp}}$$

$$\text{MotorRPM} \times \text{Motor dia.} = \text{FanRPM} \times \text{Fan dia.}$$

$$\text{Min Fan Sheave Dia.} = \frac{\text{Current Fan Sheave Dia.}}{\sqrt[3]{\frac{\text{Max BHP}}{\text{Current Est. BHP}}}}$$

$$\frac{\text{RPM Fan}}{\text{RPM Motor}} = \frac{\text{Pitch d. Motor}}{\text{Pitch D. Fan}}$$

$$\text{Max Motor Sheave dia.} = \text{Current Motor Sheave dia.} \times \sqrt[3]{\frac{\text{MAX BHP}}{\text{Current EST. BHP}}}$$

HEAT TRANSFER

AIR: (*h=Enthalpy in BTU per lb.*)

$$\text{BTUH (Sensible)} = \text{CFM} \times 1.08 \times \Delta T$$

$$\text{BTUH (Total)} = \text{CFM} \times \Delta h \times 4.5$$

$$\text{BTUH (Sensible)} = \text{CFM} \times 1.08 \times \Delta T \times \left(\frac{\text{Den}}{.075} \right)$$

$$\text{BTUH (Total)} = \text{CFM} \times \Delta h \times 4.5 \times \left(\frac{\text{Den}}{.075} \right)$$

$$\Delta T = \frac{\text{BTUH(Sensible)}}{1.08 \times \text{CFM}}$$

$$\Delta T = \frac{\text{BTUH(Total)}}{4.5 \times \text{CFM}}$$

$$\text{CFM} = \frac{\text{BTUH(Sensible)}}{1.08 \times \Delta T \text{ (Dry Bulb)}}$$

$$\text{CFM} = \frac{\text{BTUH(Total)}}{4.5 \times \Delta h}$$

$$\text{Latent Heat Transfer} = .069 \times \text{CFM} \times \Delta \text{gr}$$

$$\text{Heat Loss in BTUH} = \text{Area} \times \Delta T \times \left(\frac{1}{R_{\text{Total}}} \right)$$